



Water Injection Effects on Compressor Stage Operation

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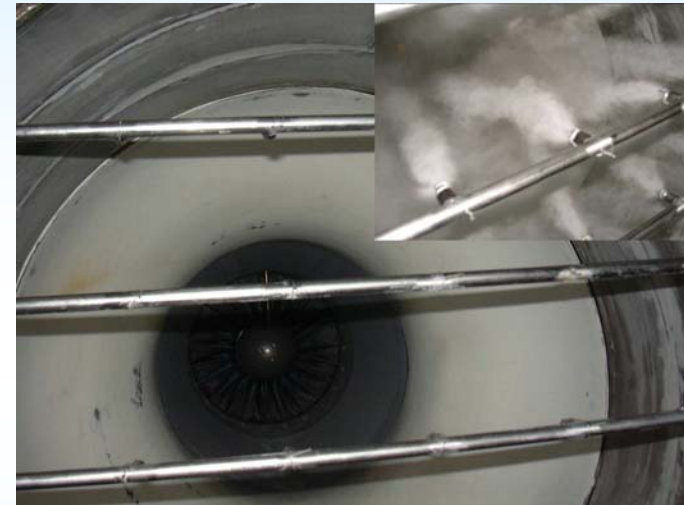
- **Experimental Set-Up**
- **Effect of Water Injection on Stage Characteristic**
- **Detailed Aerodynamic Measurements**
- **Effect on Stage Power Consumption**
- **Conclusions**



Experimental Set - Up

One Stage Axial Compressor

- $N_{C,nom} = 18000 \text{ rpm}$
- $\pi_{C,nom} = 1.57$
- $m_{C,nom} = 27.5 \text{ kg/s}$



Impaction Pin Nozzles (Mee Industries)

- **Axisymmetrically Distributed at Bellmouth Inlet**
- **Maximum Injected Water Mass Flow: 2%**

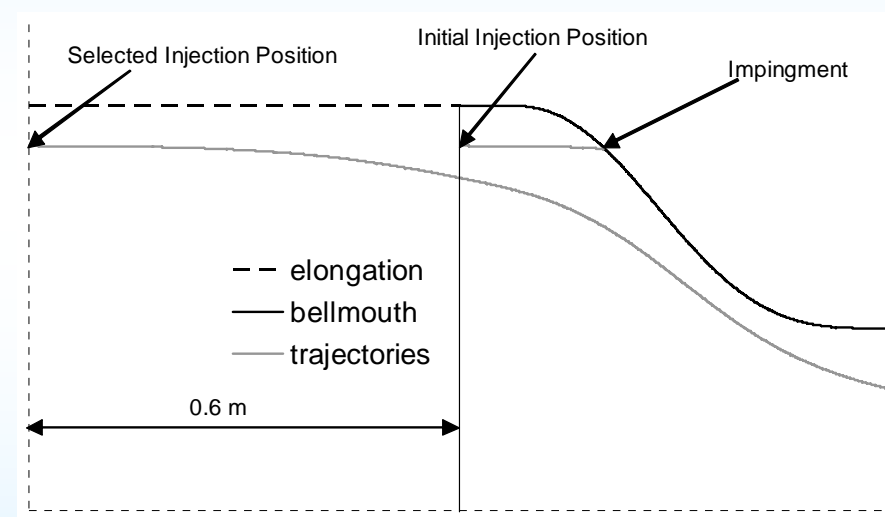


Experimental Set - Up

Bellmouth Configuration

Droplet Trajectories Analysis

- Lagrangian Framework
- Droplet Evaporation Model
- Meridional Flow Solver (2D)

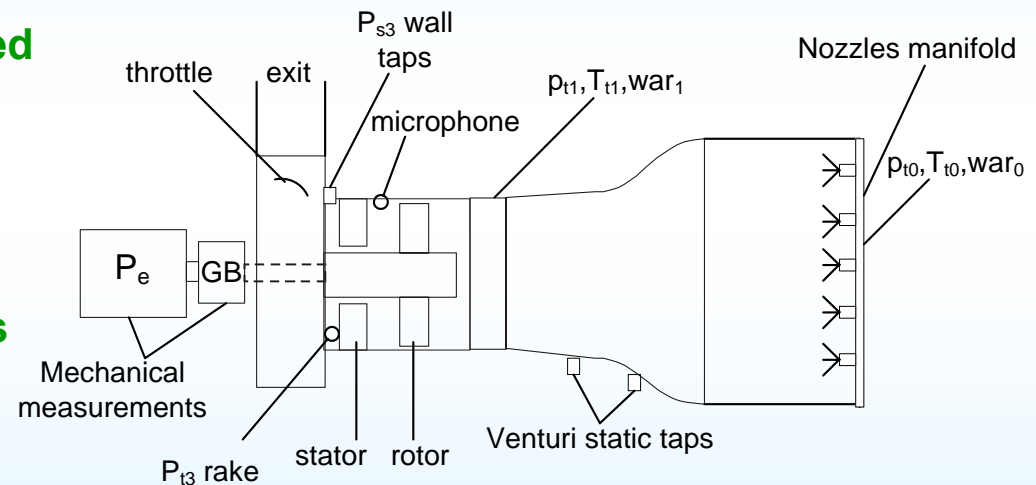




Experimental Set-Up

Test-Rig Measurements

1. The flow rate of water
2. The pressure at nozzle manifold
3. Compressor Rotational Speed
4. Torque of the Shaft
5. Ambient Conditions ($p_{t0}, T_{t0}, \omega_{ar0}$)
6. Compressor Inlet Conditions ($p_{t1}, p_{s1}, T_{t1}, \omega_{ar1}$)
7. Inlet Flow Rate (Venturi)
8. Compressor Exit Conditions (p_{t3}, p_{s3})
9. Oil Thermal Losses





Experimental Set-Up

Measurements Considerations

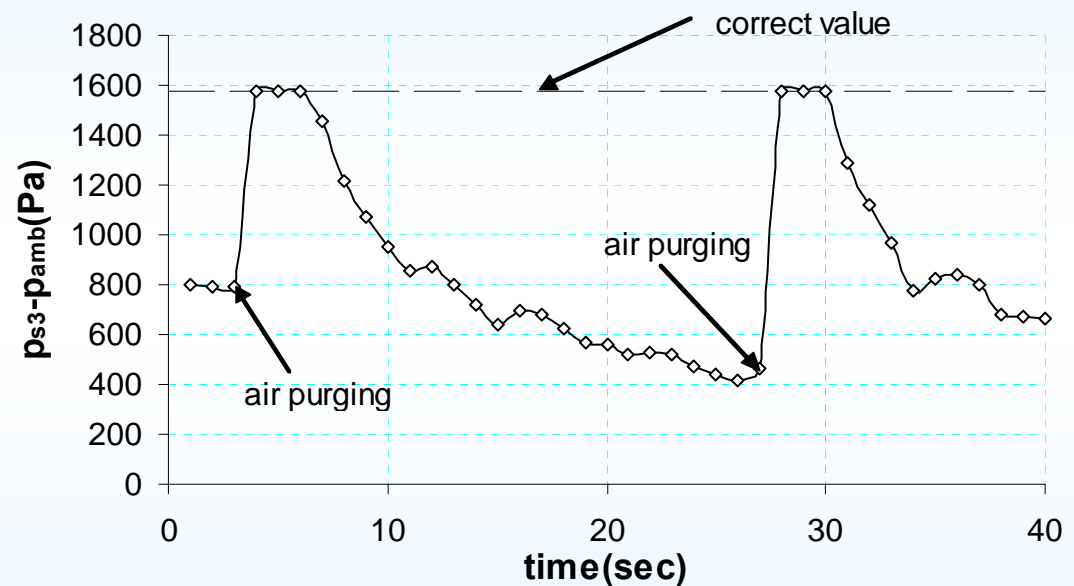
- **Pressure Measurement: Potential Plugging of Pressure Holes and Connections due to Droplets**
- **Temperature Measurement: Droplets impingement at Temperature Sensors tend to Mask the Flow Temperature**
- **Humidity Measurement: Influencing the Measurements used for calculating the Stage Performance Characteristics**
- **Unstable Operation Determination: Hot Wire use in Droplet Laden Flow is not Plausible**



Experimental Set-Up

Pressure Measurements

- Increased Measuring Wall Taps Diameter to 1.5mm
- Total Pressure Probe of Kiel Type with 3mm Hole
- Purging System using Pressurized Air and Electronic Vanes
- Defining of Acquisition Time to 1 sec

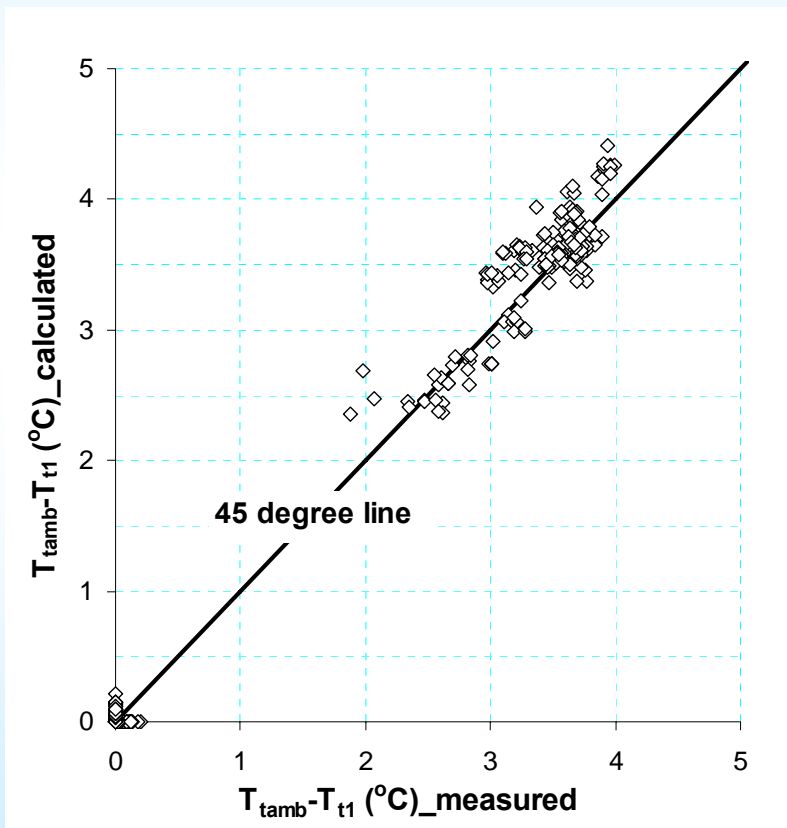




Experimental Set-Up

Inlet Water to Air Ratio and Temperature Measurement

- Bleeding Air from the Stage Inlet introducing a bent tube facing downstream
- Humidity Chamber incorporating a RH Sensor and a Pt100 Thermometer
- Measuring Temperature with a Pt100 Thermometer in the bent tube (insulated)
- Air Movement Imposing Pressure Difference at the Humidity Chamber

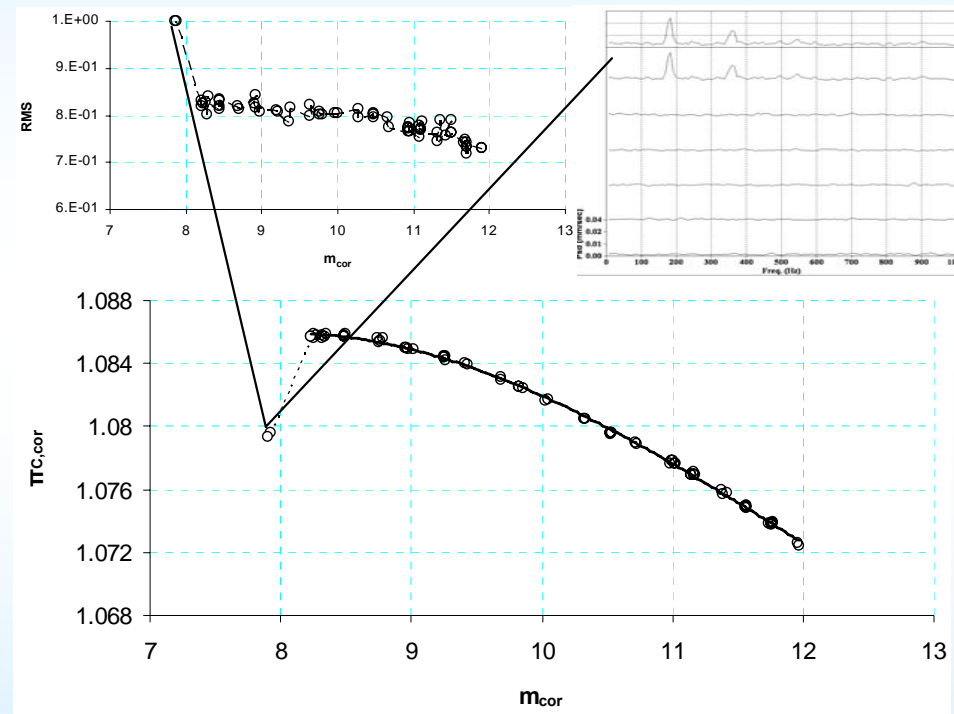




Experimental Set-Up

Unstable Operation Determination

- Unstable operation can be detected by acoustic measurements
- Use of a Hot-Wire for dry Measurements
- Microphones used for Droplet Laden Flows





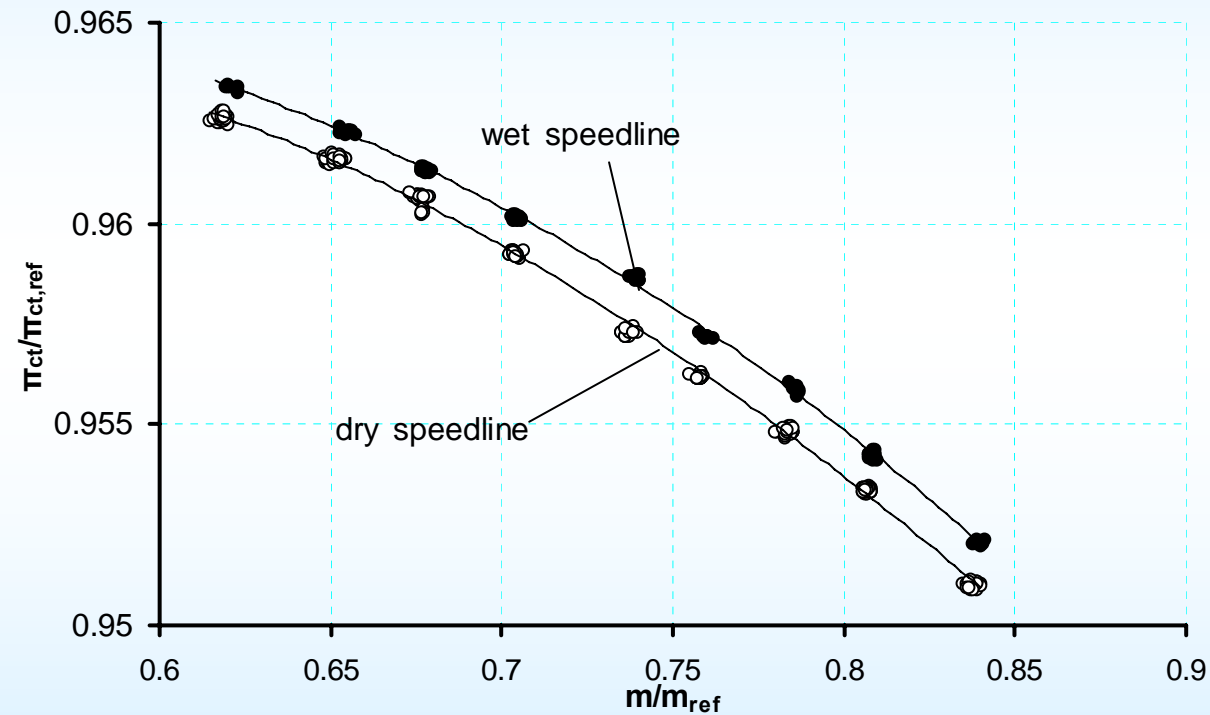
Water Injection Effect on Compressor Stage Operation

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Effect of Water Injection on Compressor Performance

Raw Effect on Compressor Performance



$T_{amb} = 20.5^{\circ}\text{C}$ RH=62% $m_{winj}=1\%$



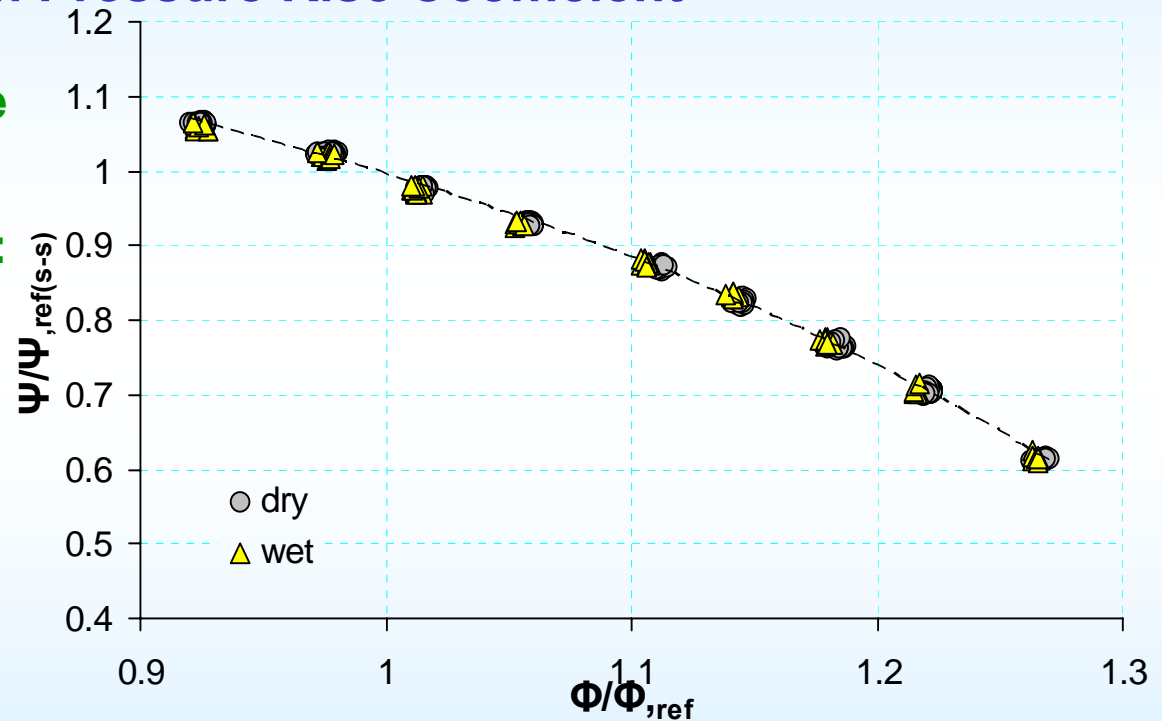
Effect of Water Injection on Compressor Performance

Effect on Pressure Rise Coefficient

- Assuming Incompressible Flow

- Pressure Rise Coefficient:

$$\Psi_{s-s, is} = \frac{p_{s3} - p_{s1}}{\rho_1 \times U_{tip}^2}$$



Tamb = 20.5°C RH=62% mwinj=1%



Effect of Water Injection on Compressor Performance

Effect on Pressure Rise Coefficient

- Assuming Incompressible Flow

- Manifold Pressure: 50bar

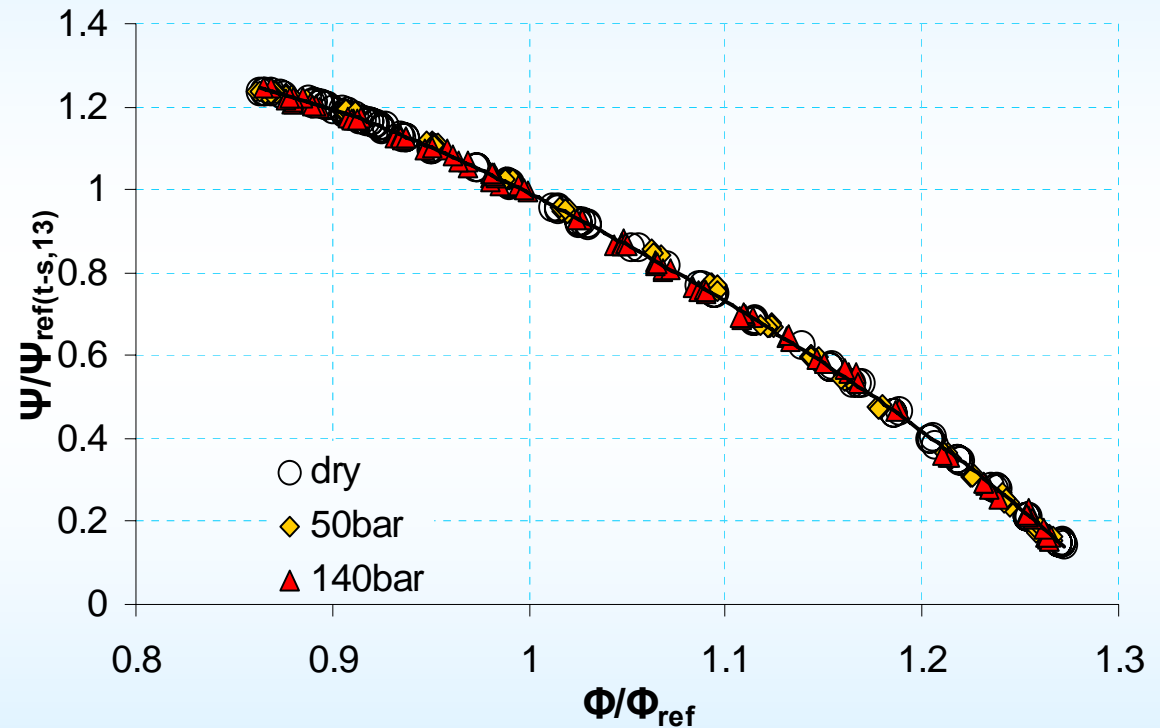
 - $m_{winj}/m_{air}=0.6\div 0.85\%$

 - MMD=20 μ m

- Manifold Pressure: 140bar

 - $m_{winj}/m_{air}=1.1\div 1.5\%$

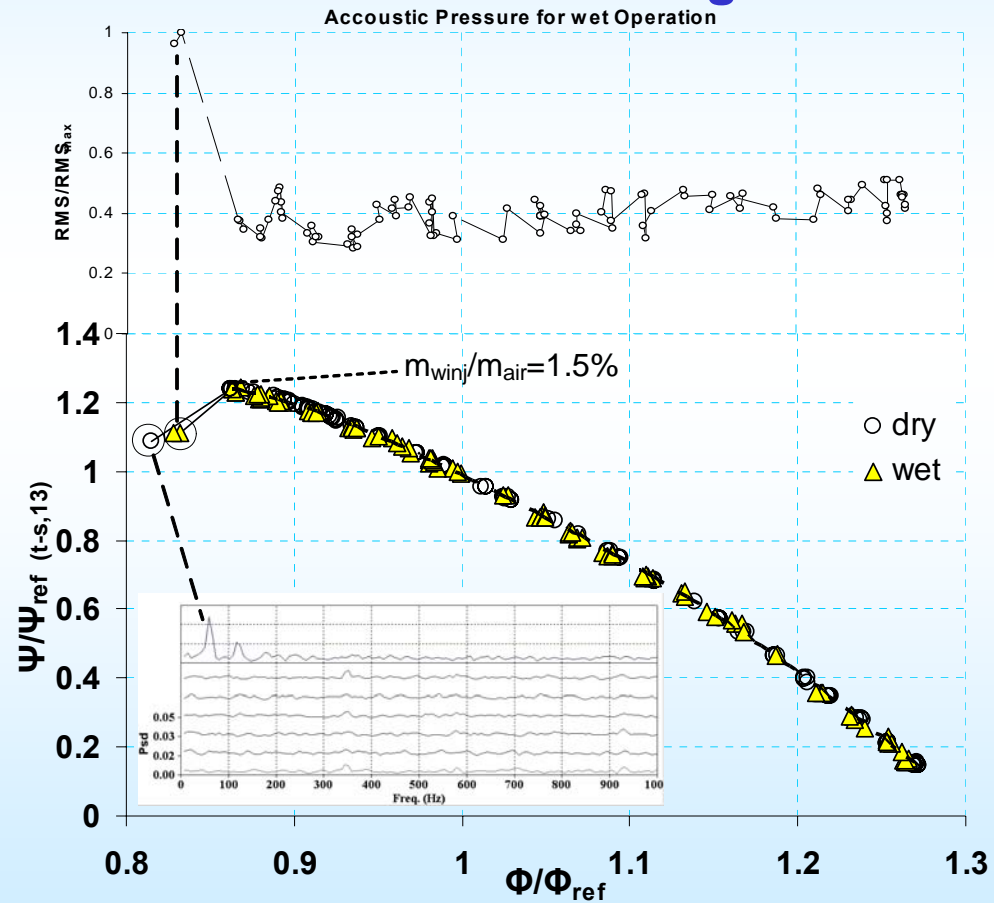
 - MMD=12 μ m





Effect of Water Injection on Compressor Performance

Effect on Stall Margin



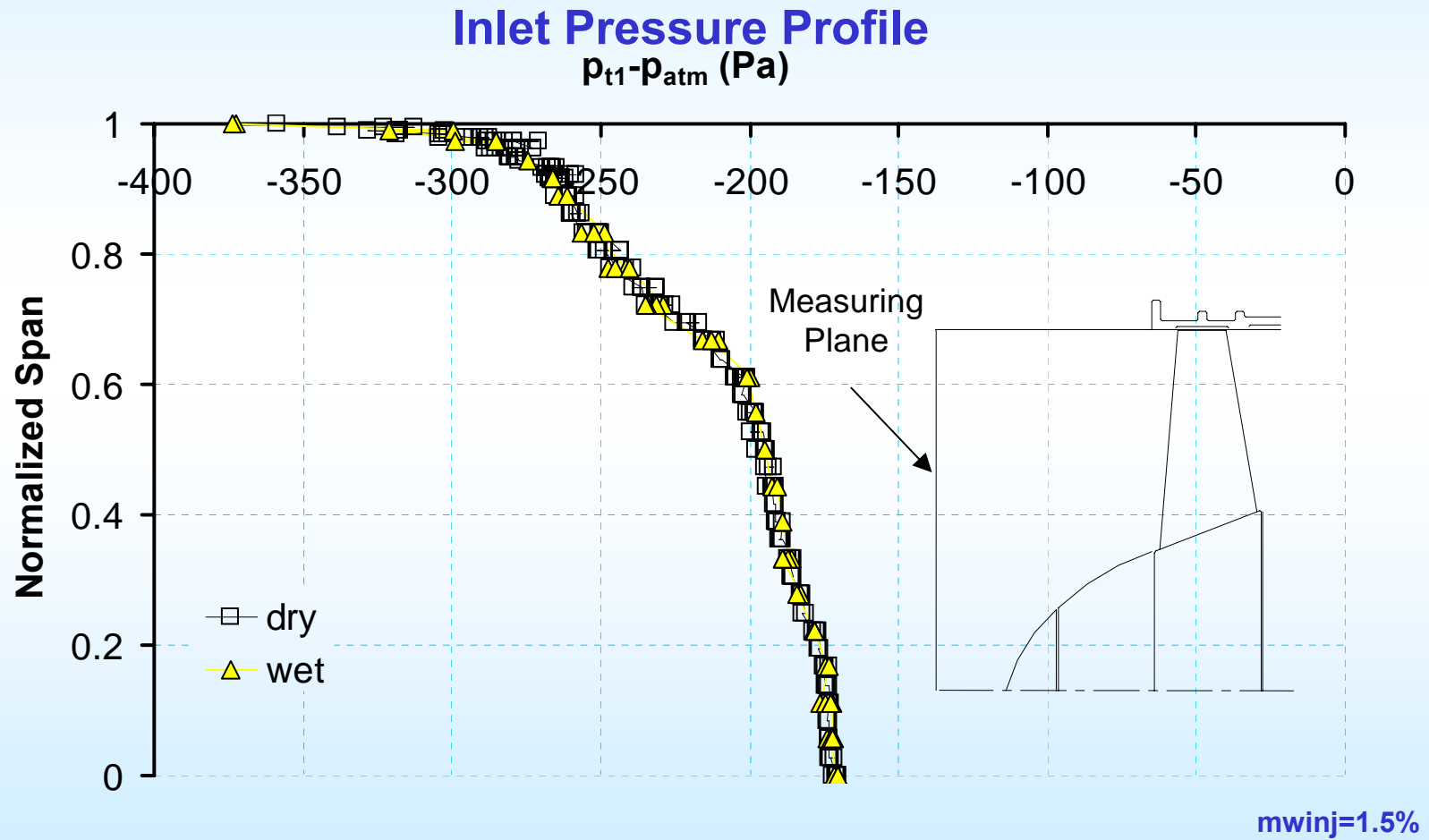


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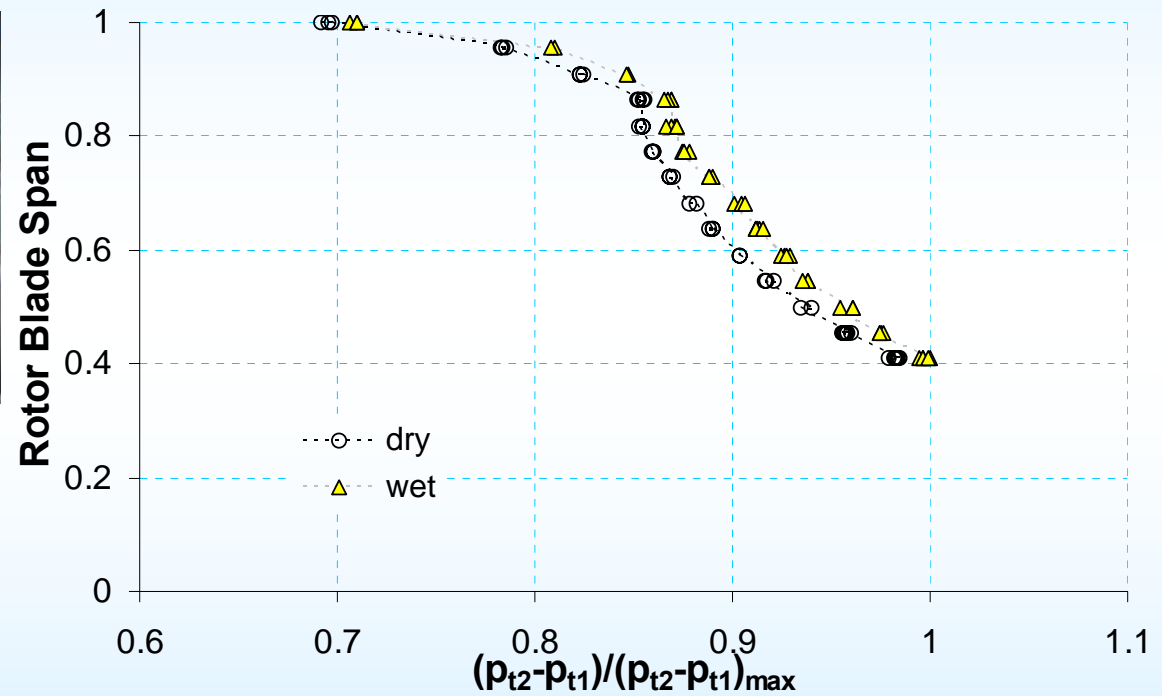
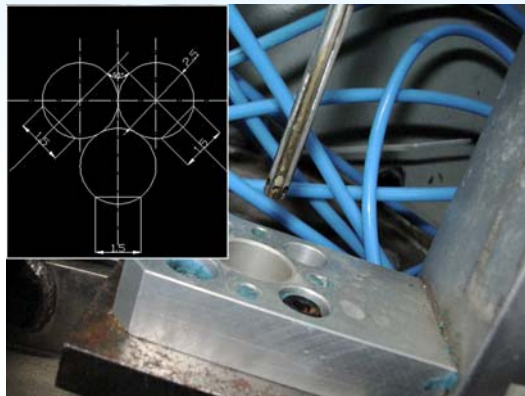
Detailed Aerodynamic Measurements





Detailed Aerodynamic Measurements

Rotor Exit Pressure Profile



$m_{inj}=2\%$



Effect of Water Injection on Compressor Performance

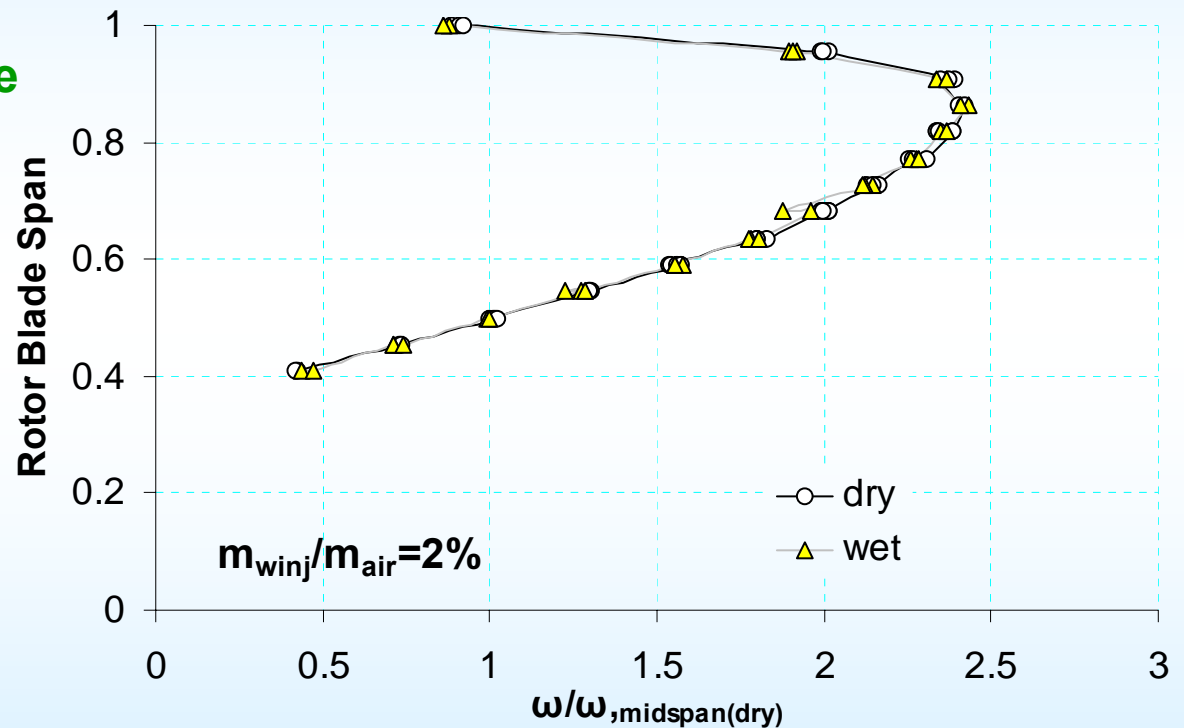
Total Pressure Loss Coefficient Profile

- Assuming Incompressible Flow

- Total Pressure Loss Coefficient:

$$\bar{\omega} = \frac{P_{tR1} - P_{tR2}}{1/2 \times \rho_1 \times W_1^2}$$

$$P_{tR} = P_s + \rho \times \frac{W^2}{2}$$





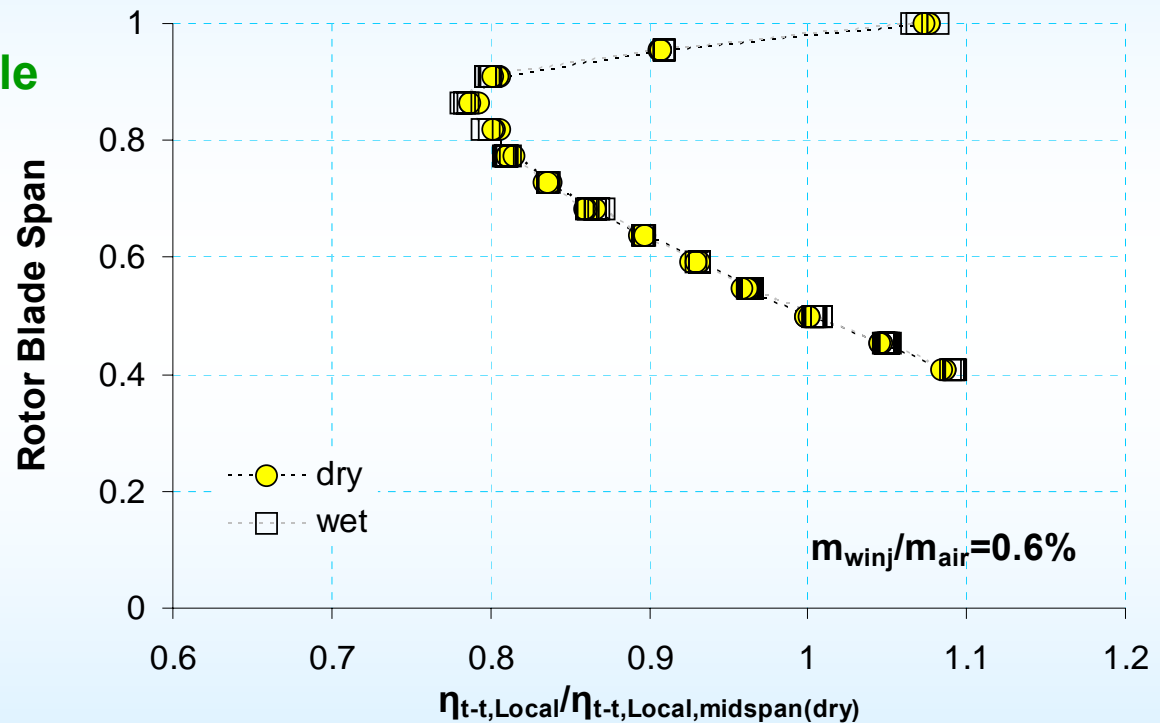
Effect of Water Injection on Compressor Performance

Total to Total Efficiency Profile

- Assuming Incompressible Flow

- Local Total to Total Efficiency:

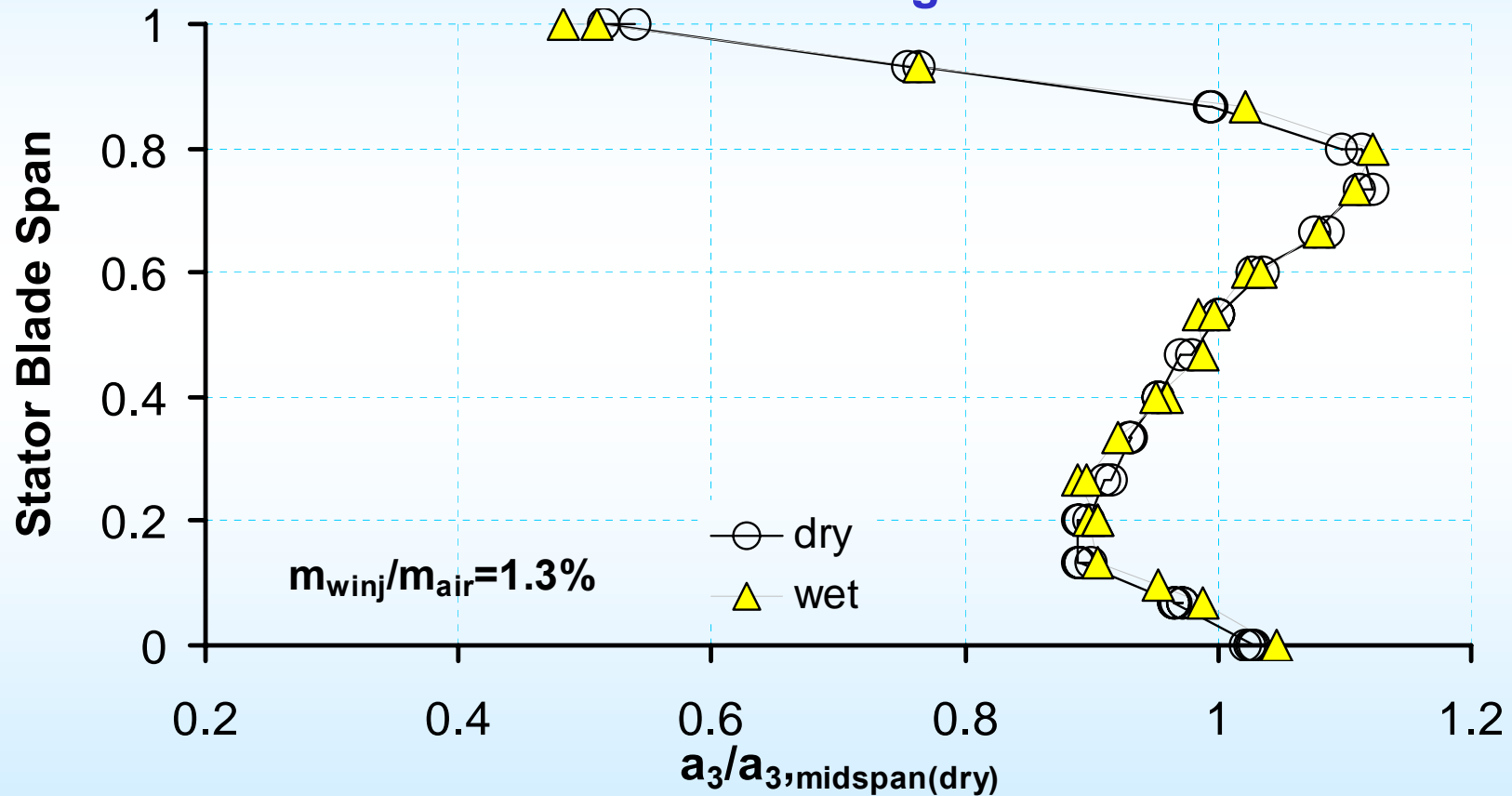
$$\eta_{t-t,Local} = \frac{p_{t2} - p_{t1}}{\rho_1 \times U (V_{u2} - V_{u1})}$$





Detailed Aerodynamic Measurements

Stator Absolute Exit Angle Profile



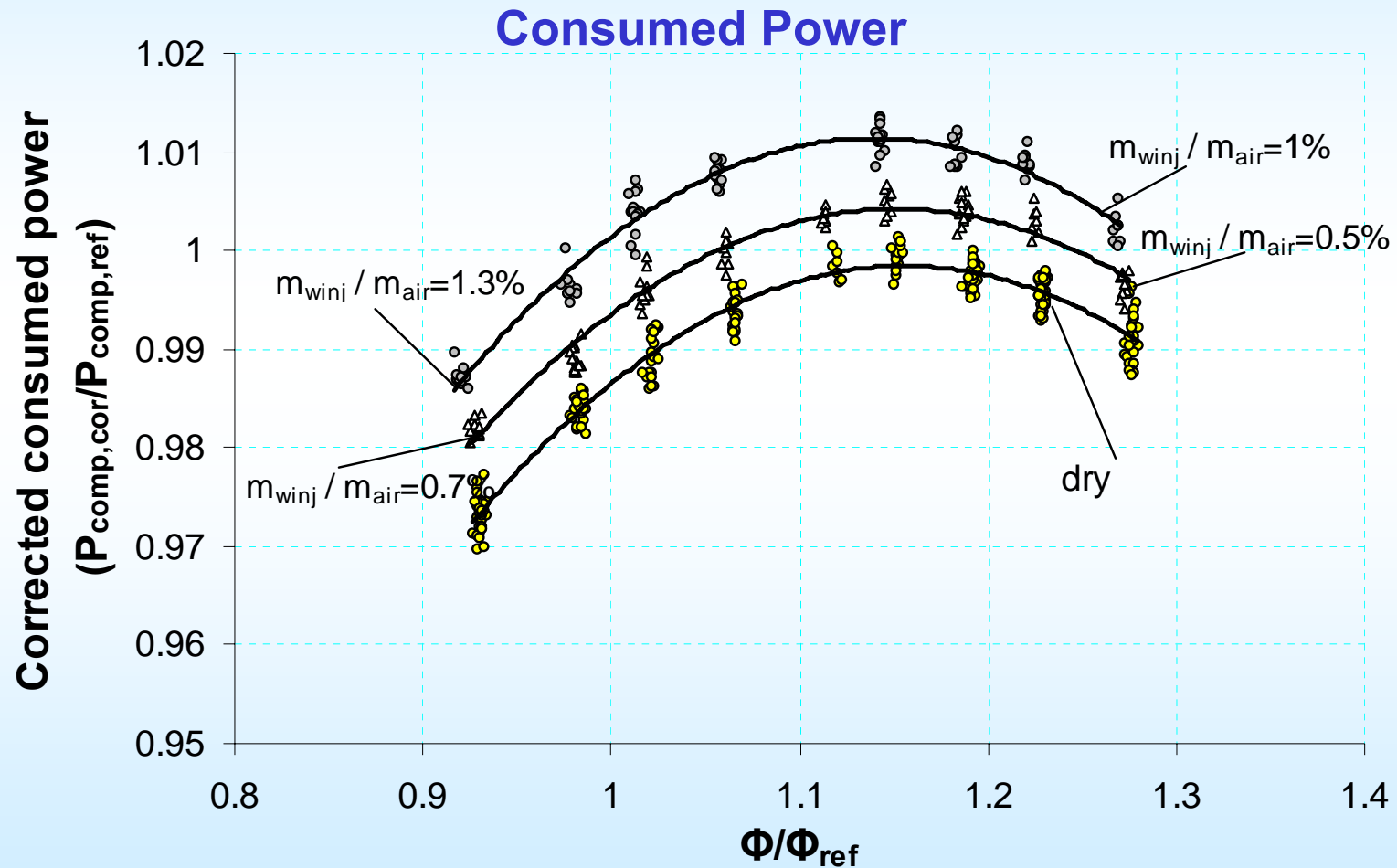


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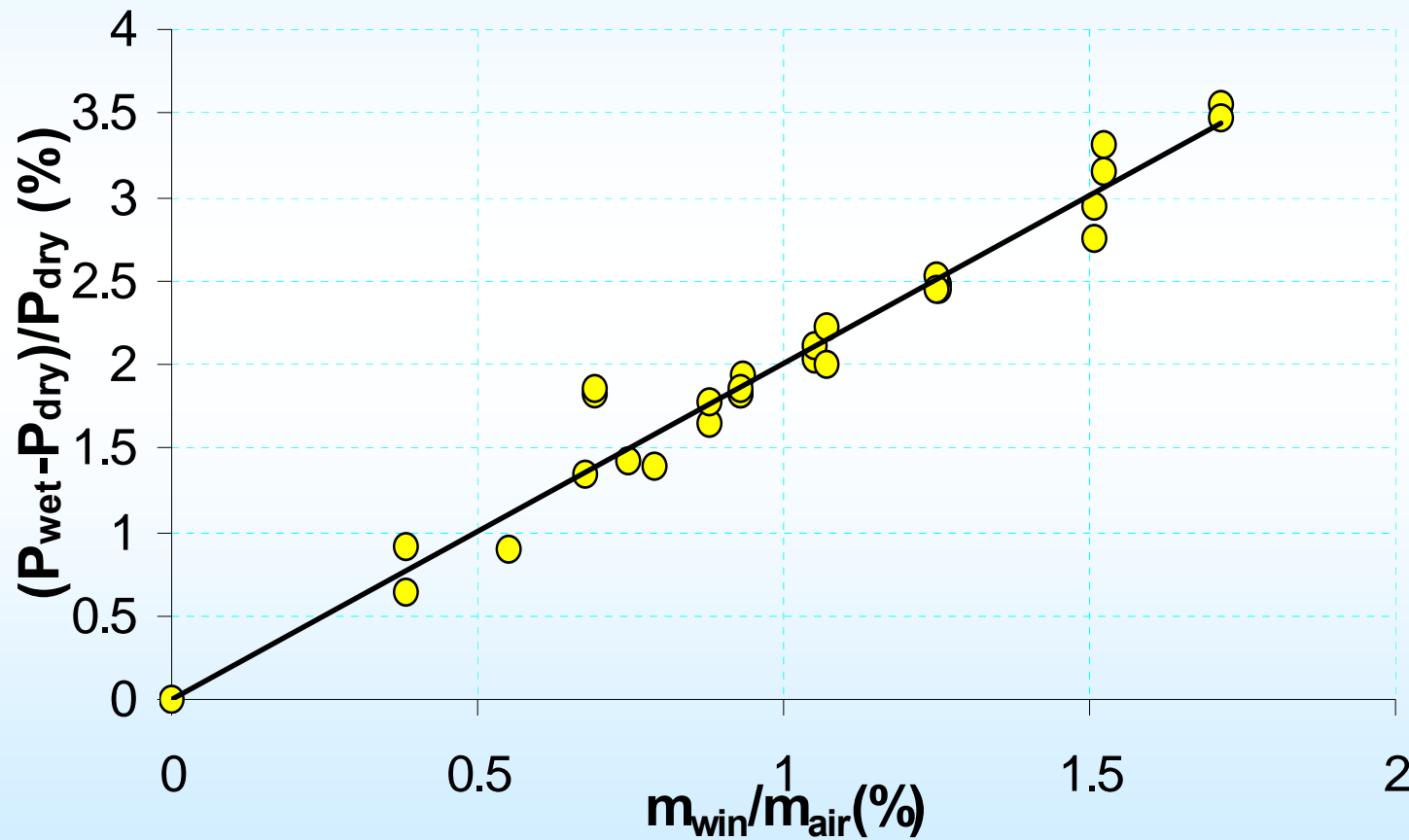
Effect on Compressor Power Consumption





Effect on Compressor Power Consumption

Power Increase



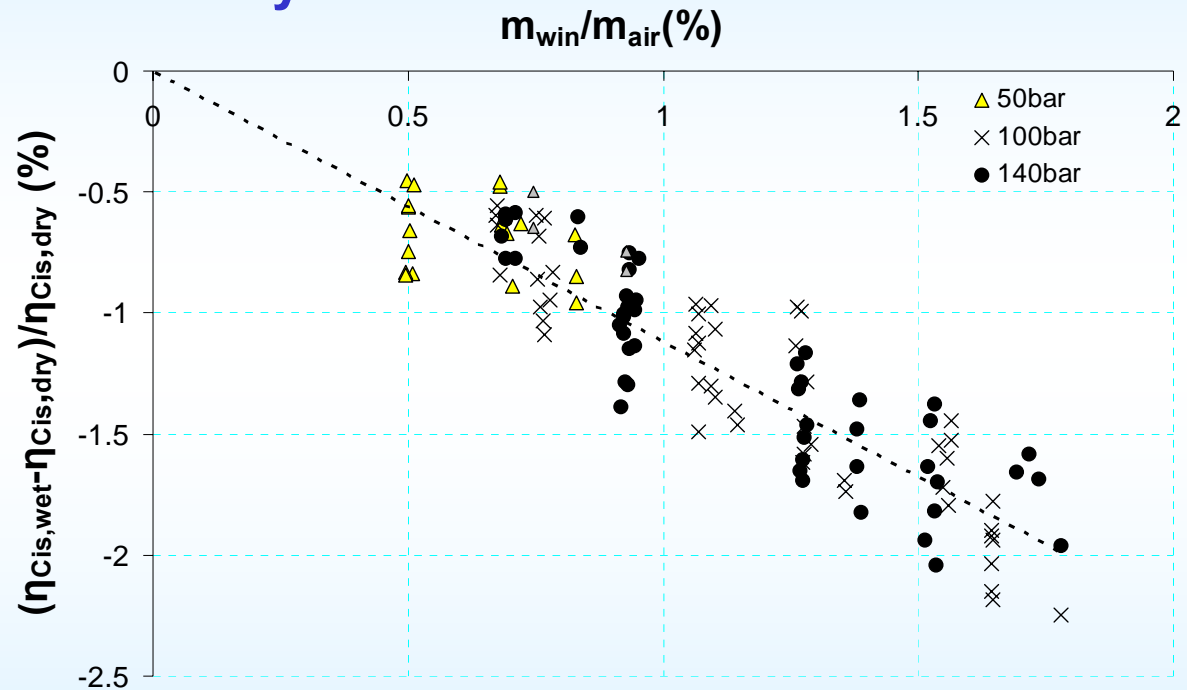


Effect on Compressor Efficiency

Efficiency Decrease

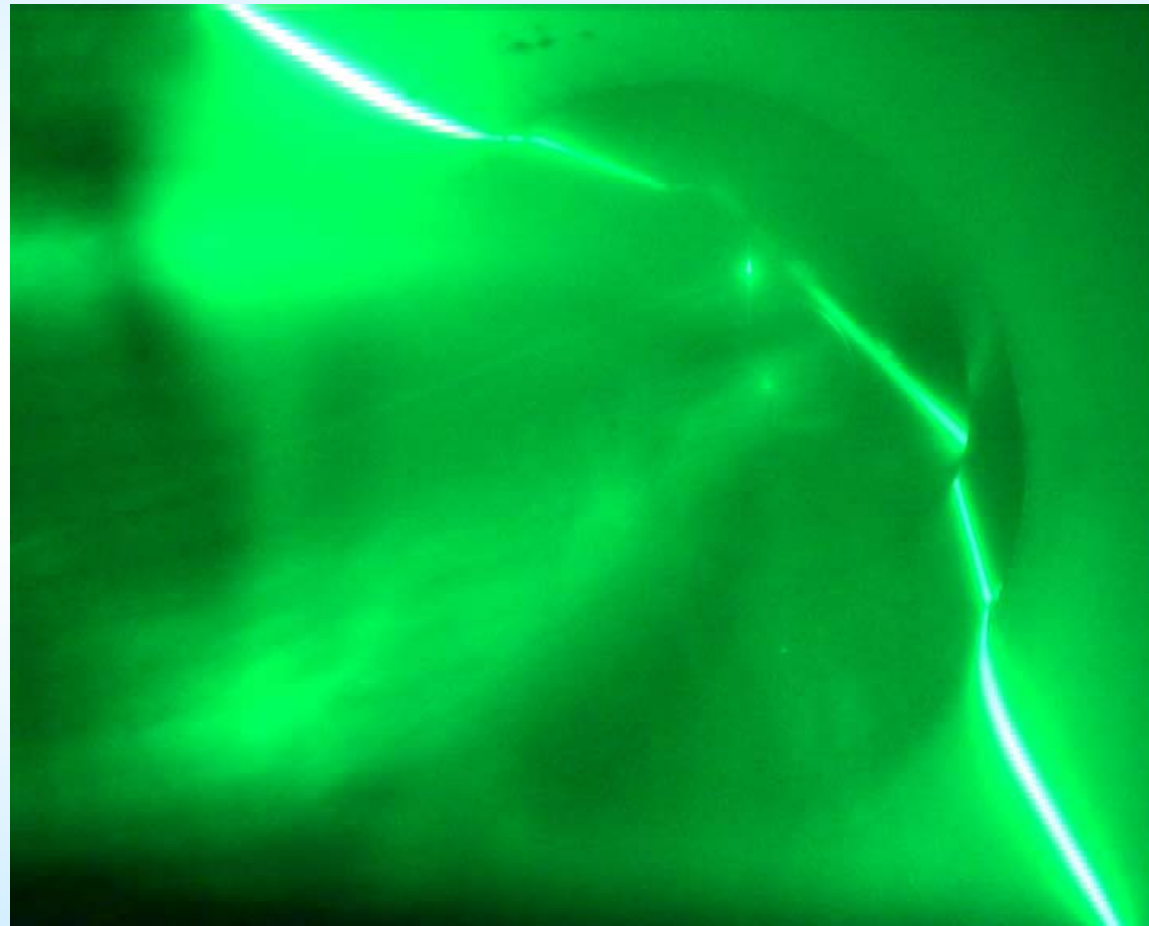
- Assuming Incompressible Flow
- Isentropic Efficiency via Mechanical Measurements :

$$\eta_{C,t-t,is} = \frac{\Delta p_{t-t,13}/\rho_1}{P_{COMP}/m_{in}}$$





Preliminary Flow Visualization





Conclusions

- Special Techniques should be used in order to obtain data from a Wet Compressor
- The pressure rise coefficient presented no measurable deviation with water injection up to 2%
- The point where initial stall occurs presented no change with droplet laden flow
- Aerodynamic measurements indicate that the stage flowfield will not be affected by the presence of droplets for water quantities up to 2%
- Water injection result to an increase of power consumption
- The decrease of stage efficiency is a strong function of the water quantity entering the engine and droplet size (from 12 μ m to 20 μ m) seems to have no effect



Future Work

- **Further examination of the mechanism resulting to the increase of consumed power is needed**
- **Visual examination of droplet behavior at rotor and stator**
- **Quantification of the losses in correlation to water collection rate on rotor blades**
- **Examination of the effect of injection position at power consumption**



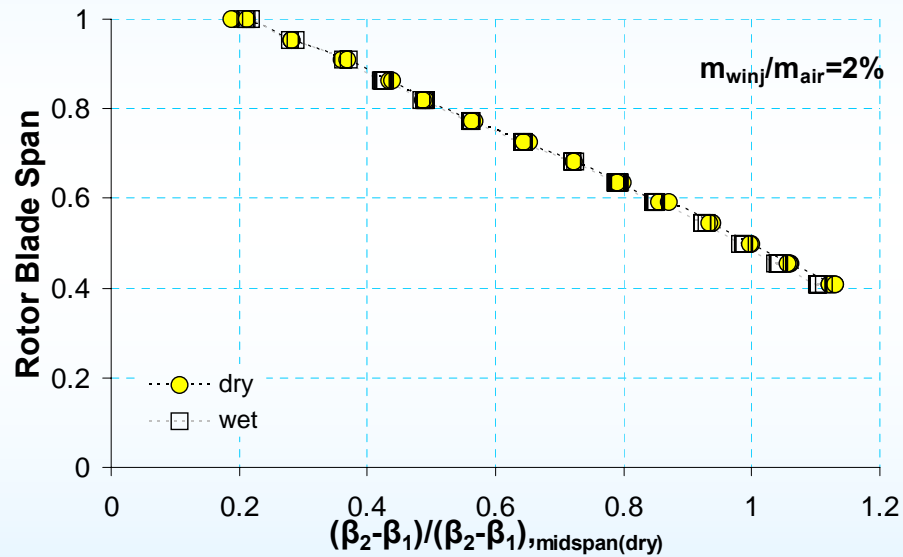
Acknowledgments

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- The cooperation of Mee Industries and Mr T. Mee personally in purchasing the nozzle manifold is gratefully acknowledged

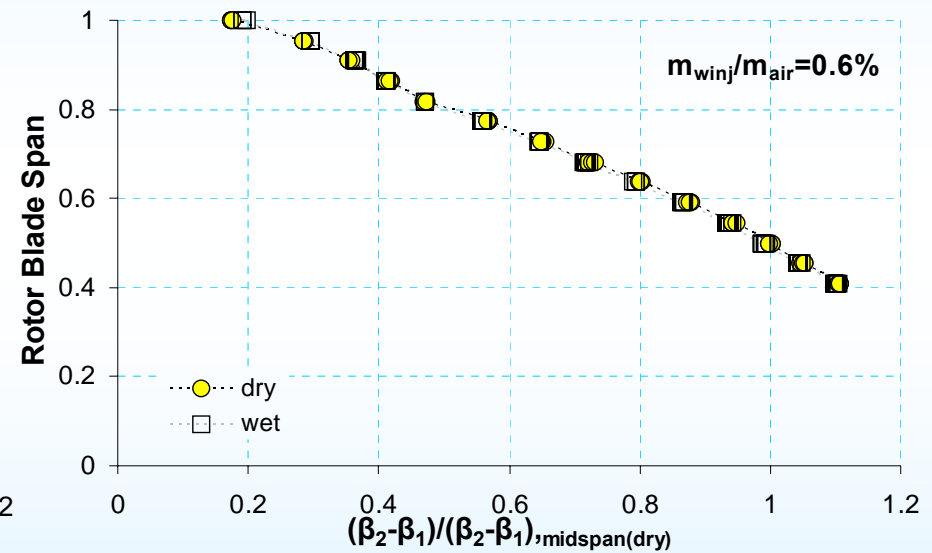


Effect of Water Injection on Compressor Performance

Rotor Relative Angle Turn Profile



$N/N_d = 27\%$



$N/N_d = 50\%$